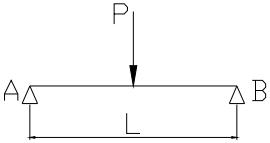
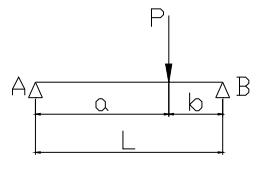
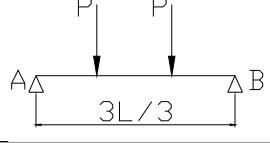
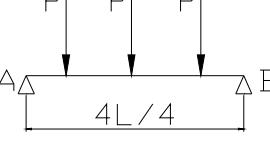
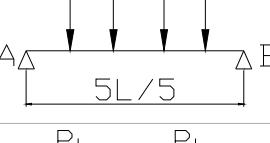
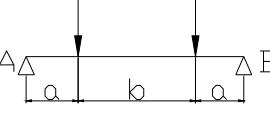
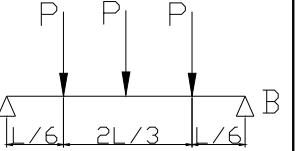
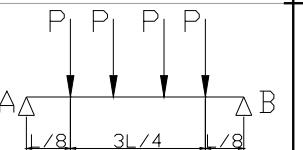
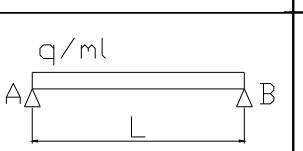
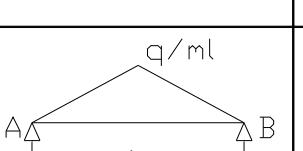
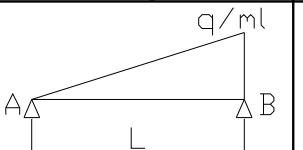
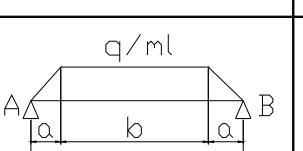
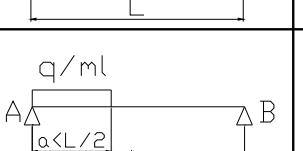
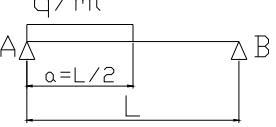
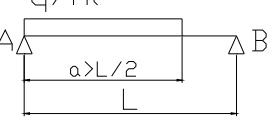
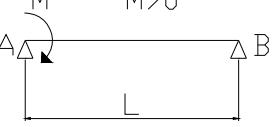
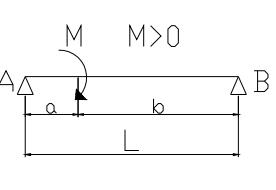
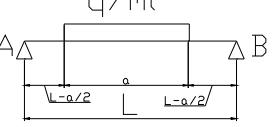
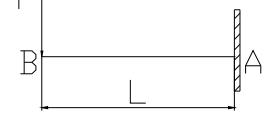
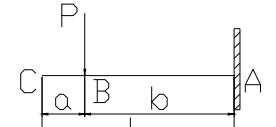
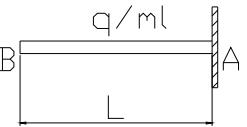
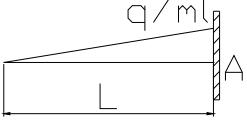


FORMULAIRE DES POUTRES

Cas de charges	Réactions aux appuis	Moment maximum	flèche L en m H en mm σ en DaN/mm ²	Flèche à l/2	Rotation aux appuis
	$\frac{P}{2}$	$M_{L/2} = \frac{PL}{4}$	$0.79\frac{\sigma L^2}{h}$	$\frac{PL^3}{48EI}$	$\theta_A = -\frac{PL^2}{16EI}$ $\theta_B = +\frac{PL^2}{16EI}$
	$R_A = \frac{Pb}{L}$ $R_B = \frac{Pa}{L}$	$M_0 = M_a = \frac{Pab}{L}$ $M_{L/2} = \frac{Pb}{2}$ ($a > b$)		$f_{l/2} = \frac{-Pb}{48EI} (3L^2 - 4b^2)$ $f_a = \frac{-P a^2 b^2}{3EI L}$ $f_{max} = \frac{-Pb}{27EI L} \sqrt{3(L^2 - b^2)^3}$	$\theta_A = \frac{Pb}{6EI L} (b^2 - L^2)$ $\theta_B = \frac{Pa}{6EI L} (L^2 - a^2)$
	P	$M_{L/2} = \frac{PL}{3}$	$1.01\frac{\sigma L^2}{h}$	$\frac{23PL^3}{648EI}$	
	$\frac{3P}{2}$	$M_{L/2} = \frac{PL}{2}$	$0.84\frac{\sigma L^2}{h}$	$\frac{19PL^3}{384EI}$	
	$2P$	$M_{L/2} = \frac{3PL}{5}$	$1.0\frac{\sigma L^2}{h}$	$\frac{63PL^3}{1000EI}$	
	P	$M_{L/2} = Pa$	$\frac{\sigma L^2}{h}$	$\frac{Pa(3L^2 - 4a^2)}{24EI}$	

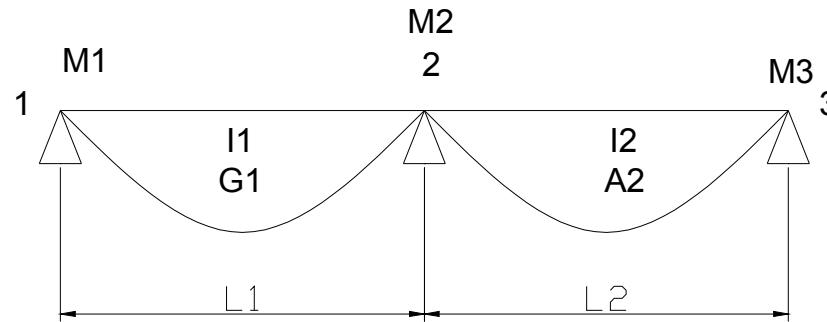
	$\frac{3P}{2}$	$M_{L/2} = \frac{5PL}{12}$	$0.94 \frac{\sigma L^2}{h}$	$\frac{53PL^3}{1296EI}$	
	$2P$	$M_{L/2} = \frac{PL}{2}$	$0.94 \frac{\sigma L^2}{h}$	$\frac{41PL^3}{768EI}$	
	$\frac{qL}{2}$	$\frac{qL^2}{8}$	$0.99 \frac{\sigma L^2}{h}$	$\frac{5qL^4}{384EI}$	$\theta_A = -\frac{qL^3}{24EI}$ $\theta_B = +\frac{qL^3}{24EI}$
	$\frac{qL}{4}$	$\frac{qL^2}{12}$	$0.95 \frac{\sigma L^2}{h}$	$\frac{qL^4}{120EI}$	$\theta_A = -\frac{5qL^3}{192EI}$ $\theta_B = +\frac{5qL^3}{192EI}$
Cas de charges multiples			$\approx \frac{\sigma L^2}{h}$		
	$RA = \frac{qL}{6}$ $RB = \frac{qL}{3}$	$M_0 = \frac{qL^2 \sqrt{3}}{27}$ $M_{L/2} = \frac{qL^2}{16}$		$f_{L/2} = -\frac{5qL^4}{768EI}$ $f_{\max} = -\frac{5qL^4}{765EI}$	$\theta_A = -\frac{7qL^3}{360EI}$ $\theta_B = +\frac{8qL^3}{360EI}$
	$RA = \frac{q}{2}(a+b)$ $RB = \frac{q}{2}(a+b)$	$M_0 = M_{L/2} = \frac{q}{24}(3L^2 - 4a^2)$		$f_{\max} = f_{L/2} = -\frac{q}{EI} \left(\frac{a^2 L^2}{48} + \frac{a^4}{120} - \frac{5L^4}{384} \right)$	$\theta_A = +\frac{q}{24EI} (2a^2 L - a^3 L^3)$ $\theta_B = +\frac{q}{24EI} (L^3 + a^3 - 2a^2 L)$
	$RA = \frac{qa}{L} \left(L - \frac{a}{2} \right)$	$M_x = \frac{L/2}{0} = RAx - \frac{qx^2}{2}$		$f_{L/2} = -\frac{qa^2}{96EI} (2a^2 - 3L^2)$	

	$R_B = \frac{qa^2}{2L}$ $M_x \frac{L}{L/2} = R_A x - \frac{qa}{2} \left(x - \frac{a}{2} \right)$		$f_{L/2} = -\frac{5qL^4}{768EI}$	
			$f_{L/2} = -\frac{q}{48EI} \left[\frac{L^4}{16} + \left(a(2L-a) - \frac{L^2}{4} \right)^2 \right]$	
	$R_A = -\frac{M}{L}$ $R_A = \pm \frac{M}{L}$ $M_0 = M_A = M$ $M_B = 0$		$f_{L/2} = -\frac{ML^2}{16EI}$ $f_{maxi} = -\frac{ML^2}{15.58EI}$	$\theta_A = -\frac{ML}{3EI}$ $\theta_B = +\frac{ML}{6EI}$
	$R_A = -\frac{M}{L}$ $R_A = \pm \frac{M}{L}$ $M_{aw} = -\frac{Ma}{L}$ $M_{ae} = +\frac{Mb}{L}$		$f_a = +\frac{Mab}{3EI} (a-b)$ $f_{L/2} = +\frac{M}{16EI} (4a^2 - L^2)$	$\theta_A = +\frac{M}{EI} \left(a - \frac{L}{3} - \frac{a^2}{2L} \right)$ $\theta_B = -\frac{M}{EI} \left(\frac{L}{6} - \frac{a^2}{2L} \right)$
	$R_A = R_B = \frac{Pa}{2}$ $M_m = +\frac{Pa}{8} (2L-a)$		$f_{L/2} = \frac{Pa}{384EI} (8L^3 - 4a^2L + a^3)$	
	$R_A = P$ $M_A = -PL$		$f_B = -\frac{PL^3}{3EI}$	$\theta_B = +\frac{PL^2}{2EI}$
	$R_A = P$ $M_A = -Pb$		$f_B = -\frac{Pb^3}{3EI}$ $f_C = -\frac{Pb^2}{6EI} (2L+a)$	$\theta_B = \theta_C = +\frac{Pb^2}{2EI}$

	$R_A = qL$	$M_A = -\frac{qL^2}{2}$		$f_B = -\frac{qL^4}{8EI}$	$\theta_B = +\frac{qL^3}{6EI}$
	$R_A = \frac{qL}{2}$	$M_A = -\frac{qL^2}{6}$		$f_B = -\frac{qL^4}{30EI}$	$\theta_B = +\frac{qL^3}{34EI}$
	$R_A = 0$	$M_A = M$		$f_B = -\frac{ML^2}{2EI}$	$\theta_B = \frac{ML}{EI}$

METHODE DE CLAPEYRON

Applicable à une poutre de module d'élasticité longitudinal constant.



$$\frac{M_1L_1}{I_1} + 2M_2\left(\frac{L_1}{I_1} + \frac{L_2}{I_2}\right) + \frac{M_3L_2}{I_2} = -6\left(\sum \frac{A_1G_1}{L_1I_1} + \sum \frac{A_2G_2}{L_2I_2}\right)$$

M1, M2, M3 moments fléchissant aux appuis

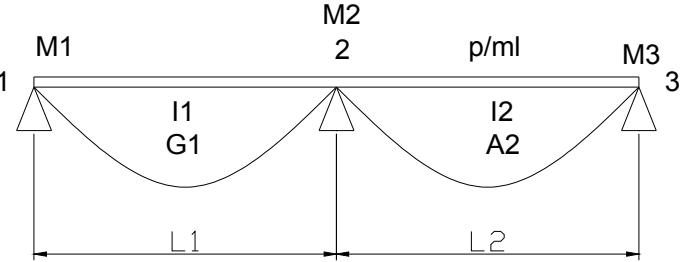
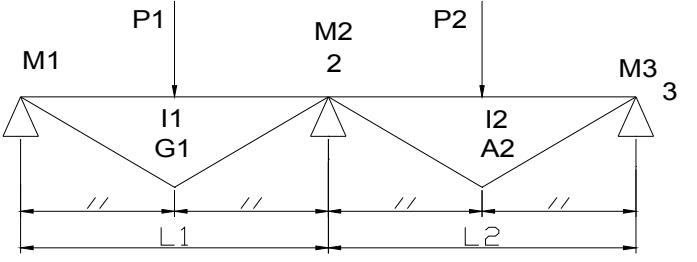
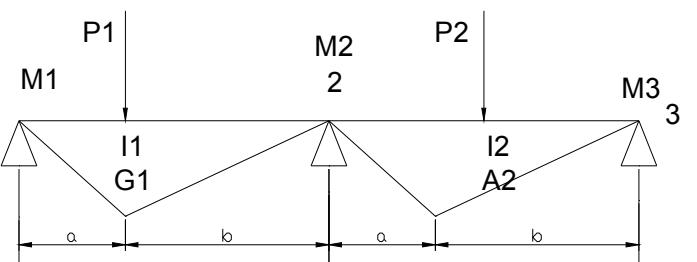
L1, L2 longueurs des travées

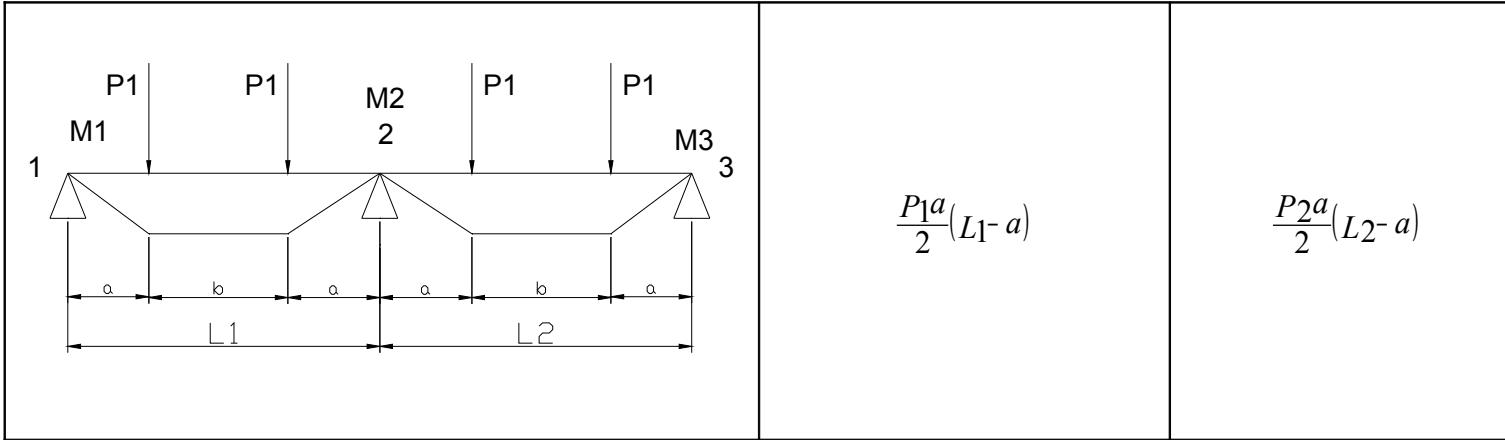
I1, I2 moments d'inerties des travées

A1, A2 aires des moments fléchissant

G1, G2 positions des centres de gravité des moments fléchissant

	A1G1/L1	A2G2/L2
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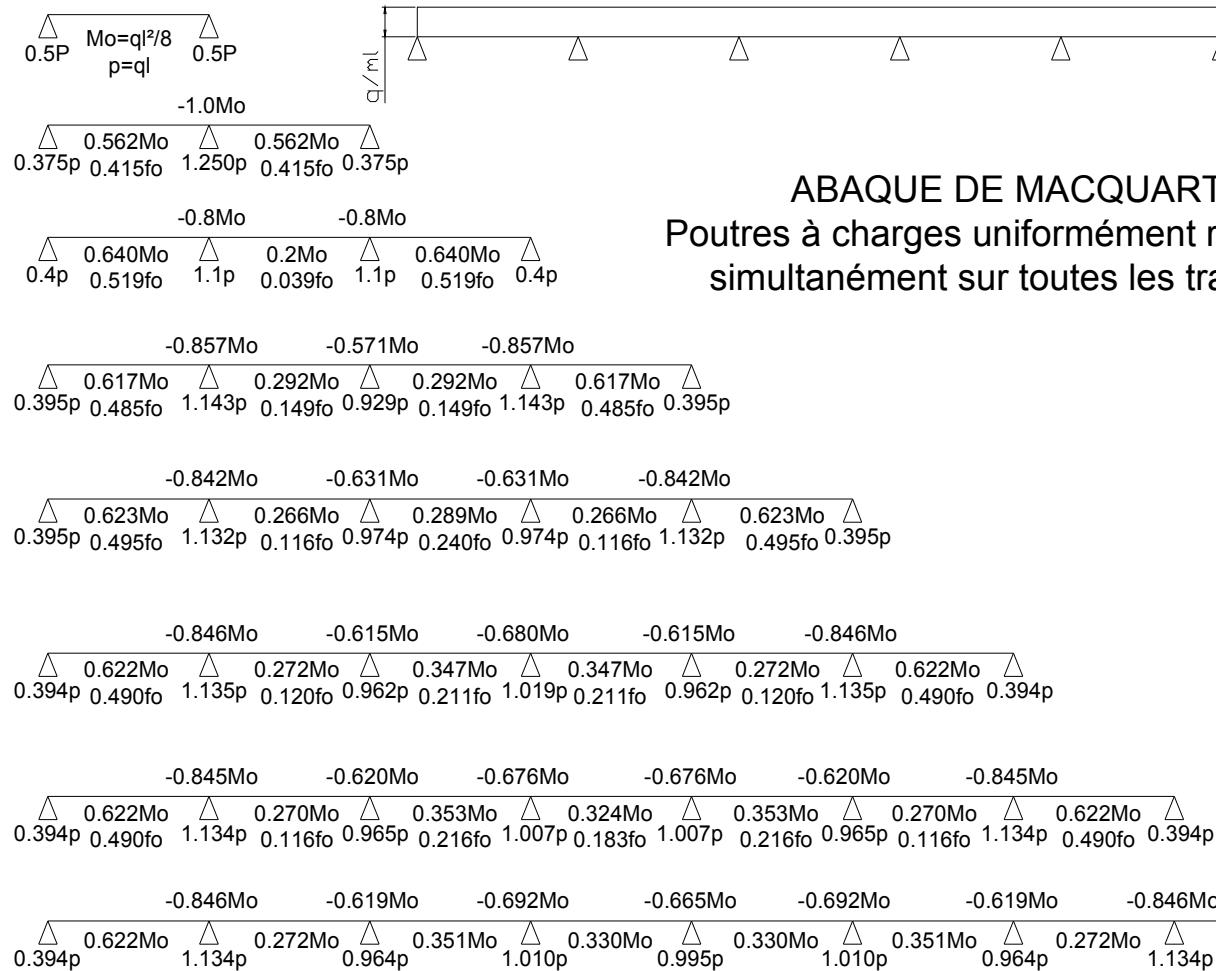
	$\frac{q_1 L_1^3}{24}$	$\frac{q_2 L_2^3}{24}$
	$\frac{P_1 L_1^2}{16}$	$\frac{P_2 L_2^2}{16}$
	$\frac{P_1 ab}{6L_1} (L_1 + a)$	$\frac{P_2 ab}{6L_2} (L_2 + b)$



$$\frac{P_1 a}{2} (L_1 - a)$$

$$\frac{P_2 a}{2} (L_2 - a)$$

ABAQUE DE MACQUART



ABAQUE DE MACQUART
Poutres à charges uniformément réparties
simultanément sur toutes les travées

dans cette abaque on calcule le moment maximum Mo, les réactions et la flèche maximum de la travée simple considérée comme isostatique, puis on applique les coefficients donnés ci-dessus pour trouver les différents moments, flèches et réactions des poutres hyperstatiques

nota : le chargement est considéré comme une CUR uniformément répartie sur toute la longueur.